AMENDMENTS TO THE SPECIFICATION:

Page 10

Please substitute the following paragraph for the paragraph beginning at line 19:

With such circumstances in mind, the present inventor first thought of ensuring the required rigidity by reducing the diameter of the balls and reducing the spacing between the balls arranged in double rows, as well as supporting the driven pulley using a double row ball bearing with reduced dimensions related to the axial direction (Japanese Patent Application No. 2002-24863, Japanese Patent Application No. 2002-97966). In the case of a pulley supporting double row ball bearing according to these related inventions, one having an outer ring with an outer diameter of less than 65 mm and a double row of outer ring raceways on the inner circumferential surface is used. Also, an inner ring having a double row of inner ring raceways on the outer circumferential surface is used. Moreover, balls with a diameter (major diameter) of less than 4 mm are used, and several of these are provided so as to roll freely between each of the outer ring raceways and each of the inner ring raceways. Also, by using a retainer, each of the balls are held so as to allow free rolling. Moreover, a pair of seal ringsring is used to seal off the openings on both sides of

the inner space accommodating each of the balls between the inner circumferential surface of the outer ring and the outer circumferential surface of the inner ring.

Furthermore, the spacing between the balls, and the spacing between the balls and the seal ring are reduced, thus providing a double row ball bearing with an overall width in the axial direction (approximately coinciding with the outer ring width and inner ring width) of less than 45% of the inner diameter of this inner ring.

Page 12

Please substitute the following paragraph for the paragraph beginning at line 2:

Also, in order to reduce the spacing between the balls, a crown shaped retainer made of synthetic resin is used for each of the retainers, and rims of each of the retainers are provided to oppose each other from opposite sides (<u>i.e.</u>[[=]] outsides in the axial direction, and [[=]] sides opposed to the seal ring). Also, the distance between the rim of each of the retainers and the inside surface of the seal ring is reduced. However, again in this case, the distance between the rim of each of the retainers and the inside surface of each seal ring is ensured to be over 13% of the diameter of each of the balls such that the filling amount of the grease

within the inner space accommodating each of the balls, between both of the seal rings can be ensured.

Page 13

Please substitute the following paragraph for the paragraph beginning at line 15:

Particularly, in a first aspect of the pulley support double row ball bearing of this invention, a portion near an inner circumference of the respective seal rings and both end surfaces in the axial direction of the inner ring overlap when viewed from the axial direction, so that a width in the radial direction of an overlap section is 25% or more than a diameter of the respective balls. Also, of a plurality of protrusions that are formed all around a circumference on an inside surface at a portion near an inner circumference of the respective seal rings, and a tip edge of at least one of the protrusions comes in sliding contact with the end surfaces in the axial direction of the inner ring.

Page 14

Please substitute the following paragraph for the paragraph beginning at line 4:

Moreover, in a second aspect of the pulley support double row ball bearing of this invention, a portion near an inner circumference of the respective seal rings and both end surfaces in the axial direction of the inner ring overlap when viewed from the axial direction, so that a width in the radial direction of an overlap section is 25% or more than a diameter of the respective balls. Also, and of one or more protrusions that are formed all around a circumference on a side surface at a portion near an inner circumference of the respective seal rings, and a tip edge of at least one of the protrusions comes in sliding contact all the way around the circumference with a part of the surface of the inner ring. Together with this, anotherthe other portion near the inner circumference of the respective seal rings, that are not being a the protrusions being in sliding contact, comes close to and faces the other part of the surface of the inner ring, so that labyrinth seals are formed.

Page 17

Please substitute the following paragraph for the paragraph beginning at line 16:

Fig. 6 is a cross-sectional view showing an example of a prior knownknow compressor.